

## Variants of projection-based finite element variational multiscale methods for the simulation of turbulent flows

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### SUMMARY

Some variants of a three-scale projection-based finite element variational multiscale (VMS) method are studied for turbulent channel flow computations at  $Re_\tau = 180$ . Different spaces for the large scales, two eddy viscosity models and two ways of discretizing the projection terms in time are explored. The results obtained with the resolved small scales in the definition of the eddy viscosity are very sensitive to the temporal discretization of the projection terms. The computations were performed on three grids commonly used in turbulent channel flow simulations. Copyright © 2007 John Wiley & Sons, Ltd.

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### 1. INTRODUCTION

Variational multiscale (VMS) methods are a rather new approach for simulating turbulent flows. They are based on basic ideas from [1, 2] and the first application of these ideas to turbulent flow simulations can be found in [3]. Meanwhile, different classes and realizations of VMS methods exist, e.g. see [4–9]. The main feature of VMS methods is the definition of scales by projections into function spaces. Naturally, scales smaller than the mesh width, the unresolved scales, cannot be computed. A first class of VMS methods uses a two-scale decomposition of the flow and multiscale concepts are applied to model the influence of the unresolved scales onto the resolved ones [9]. A second class is based on a three-scale decomposition, where the resolved scales are decomposed into large and small ones [8]. It is assumed that the scale separation is performed

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such that the direct influence of the unresolved scales onto the large scales can be neglected. The impact of the unresolved scales onto the resolved small scales is modeled by an eddy viscosity model. The definition of the scales by projections and the ways of taking the effect of the subgrid scales into account are important differences to classical LES methods, see [10] for a detailed discussion, and they avoid problems like commutation errors on boundaries [11, 12].

This paper considers a three-scale VMS method in the context of finite element discretizations, the so-called projection-based VMS method, described in Section 2. Recently, an assessment of this method for the benchmark problem of the turbulent channel flow at  $Re_\tau = 180$  has been started [10]. The present paper will continue these investigations and study in particular some variants of this VMS method with respect to the definition of the eddy viscosity model and the discretization of the projection terms in time.

## 2. PROJECTION-BASED THREE-SCALE FINITE ELEMENT VMS METHODS

Let  $V^h \times Q^h$  be a pair of inf-sup stable, conforming finite element spaces for velocity and pressure. In addition, let  $L^H$  be a finite dimensional space of symmetric  $d \times d$  tensor-valued functions representing a large-scale space, and a non-negative function  $\nu_T$  acting as the turbulent viscosity. The semi-discrete three-scale projection-based VMS method with parameters  $\nu_T$  and  $L^H$  then seeks  $\mathbf{u}^h : [0, T] \rightarrow V^h$ ,  $p^h : (0, T) \rightarrow Q^h$ , and  $\mathbb{G}^H : [0, T] \rightarrow L^H$  such that

$$\begin{aligned} & (\mathbf{u}_t^h, \mathbf{v}^h) + (2\nu \mathbb{D}(\mathbf{u}^h), \mathbb{D}(\mathbf{v}^h)) + ((\mathbf{u}^h \cdot \nabla) \mathbf{u}^h, \mathbf{v}^h) \\ & - (p^h, \nabla \cdot \mathbf{v}^h) + (\nu_T (\mathbb{D}(\mathbf{u}^h) - \mathbb{G}^H), \mathbb{D}(\mathbf{v}^h)) = (\mathbf{f}, \mathbf{v}^h) \quad \forall \mathbf{v}^h \in V^h \\ & (q^h, \nabla \cdot \mathbf{u}^h) = 0 \quad \forall q^h \in Q^h \\ & (\mathbb{D}(\mathbf{u}^h) - \mathbb{G}^H, \mathbb{L}^H) = 0 \quad \forall \mathbb{L}^H \in L^H \end{aligned} \tag{1}$$

The last equation in (1) states that  $\mathbb{G}^H$  is the  $L^2$ -projection of  $\mathbb{D}(\mathbf{u}^h)$  into  $L^H$ . Thus,  $\mathbb{G}^H$  represents large scales of  $\mathbb{D}(\mathbf{u}^h)$  and, consequently,  $\mathbb{D}(\mathbf{u}^h) - \mathbb{G}^H$  represents resolved small scales. Hence, the additional term  $(\nu_T (\mathbb{D}(\mathbf{u}^h) - \mathbb{G}^H), \mathbb{D}(\mathbf{v}^h))$ , introduced by three-scale VMS methods, is a viscous term acting directly only on the resolved small scales.

Two static Smagorinsky-type eddy viscosity models for the turbulent viscosity will be considered in the numerical tests:

- using all resolved scales:  $\nu_T = C_S \delta^2 \|\mathbb{D}(\mathbf{u}^h)\|_F$ ,
- using only the resolved small scales:  $\nu_T = C_S \delta^2 \|\mathbb{D}(\mathbf{u}^h) - \mathbb{G}^H\|_F$ ,

where  $\|\cdot\|_F$  is the Frobenius norm. Smagorinsky-type models are commonly used as turbulence models in three-scale VMS methods. Numerical studies show that using the static Smagorinsky model within VMS methods works well, often not worse than the dynamic Smagorinsky model, sometimes even better [7, 13].

All terms of (1) that occur in the Galerkin finite element formulation of the Navier–Stokes equations will be treated implicitly in time. The  $L^2$ -projection for the definition of  $\mathbb{G}^H$  will be treated explicitly as well as implicitly. The implicit discretization is described in detail in [8].

Its main features are:

- it is only efficient if  $L^H$  is a discontinuous finite element space on the finest grid and the basis of  $L^H$  is chosen to be  $L^2$ -orthogonal,
- seven sparse matrices whose dimensions depend on  $V^h$  and  $L^H$  are needed,
- four of these matrices have to be assembled only once at the initial time,
- the three other matrices have to be assembled at each discrete time in each step of the iteration for solving the nonlinearity, since they depend on  $v_T$  and  $v_T$  depends on the current finite element solution,
- matrix–matrix products have to be computed at each discrete time in each step of the iteration for solving the nonlinearity.

The explicit discretization of the projection term in the additional viscous term removes the restriction on  $L^H$  [14]. It is also somewhat cheaper than the implicit approach. Instead of the costs given in the last two points, the following arise [15]:

- the three other matrices have to be assembled only once at each discrete time,
- matrix–vector products have to be computed only once at each discrete time.

Note that the assembling of matrices in three-dimensional computations is quite expensive [16]. Altogether, one can expect faster computations with the explicit approach. The amount of speed-up will be studied in the numerical tests.

### 3. NUMERICAL RESULTS FOR THE TURBULENT CHANNEL FLOW AT $Re_\tau = 180$

The definition of the turbulent channel flow problem at  $Re_\tau = 180$  as well as references can be found in [17]. Its setup and the computation of the values of interest in our simulations have been discussed in detail in [10]. Starting with an already developed flow field, each solution was allowed to develop further for 10 s (dimensionless) and statistics were computed for another 20 s.

The computations were performed with the code *MOONMD* [18]. Hexahedral grids with  $V^h \times Q_h = Q_2 \times P_1^{\text{disc}}$  were used, i.e. the pressure finite elements are piecewise linear and discontinuous. For the temporal discretization, the Crank–Nicolson scheme was used with the equidistant time steps  $\Delta t = 0.002$  and  $0.004$ . Both time steps are smaller than the Kolmogorov time scale and they fit into the range of time steps proposed in [19]. The following variants of the projection-based VMS method will be studied:

- $L^H \in \{P_0, P_1^{\text{disc}}\}$ ; abbreviations: VMS0, VMS1;
- explicit and implicit temporal discretization of the projection term in the momentum equation and the  $L^2$ -projection, abbreviations: EXPL, IMPL;
- definition of the turbulent viscosity with all resolved scales or only with the resolved small scales, abbreviations: ALL, SMALL.

In [8, 10], only VMS0\_IMPL\_ALL and VMS1\_IMPL\_ALL were considered.

The computations were performed on grids that are uniform in streamwise and spanwise direction but which become finer toward the walls in the wall-normal direction. We studied three different types of the distribution of the grid points in wall-normal direction,  $i = 0, \dots, N_y$ , where  $N_y$  is the

number of grid points:

- Grid 1:  $y_i = 1 - \cos(i\pi/N_y)$ ,  $y_{\min}^+ = 1.7293$ , [7, 10];
- Grid 2:  $y_i = 1 + \tanh(2.75(2i/N_y - 1))/\tanh(2.75)$ ,  $y_{\min}^+ = 0.7244$ , [10, 19];
- Grid 3:  $y_i = 1 + \tanh(2(2i/N_y - 1))/\tanh(2)$ ,  $y_{\min}^+ = 2.115$ , [6].

Grid 2 of the present study is the same as Grid 0 from [10]. The degrees of freedom (d.o.f.), which are not located on mesh cell boundaries in the wall-normal direction, do not obey the prescribed distribution but are located as it is usual in finite element methods, i.e. in the center between the mesh cell boundaries for the  $Q_2$  finite element. The number  $y_{\min}^+$  is the smallest distance of a velocity d.o.f. to the wall, measured in wall units  $y^+ = Re_\tau y$ . The numerical studies were performed on  $8 \times 16 \times 8$  grids, which results in 25 344 velocity d.o.f. (including Dirichlet nodes) and 4096 pressure d.o.f. These coarse grids require the application of a turbulence model since a direct numerical simulation (DNS) blows up in a finite time [10]. The dimensions of each component of the large-scale spaces are 1024 for  $L^H = P_0$  and 4096 for  $L^H = P_1^{\text{disc}}$ . The parameter  $\delta$  in the Smagorinsky models was set to twice the length of the shortest edge of the mesh cells. This is the same choice as in [10] (where, however, in [10] erroneously only once the length of the shortest edge was given).

Two situations for the application of turbulence models can be distinguished:

- the grid is too coarse to allow a DNS, the turbulence model is needed to perform any simulations at all,
- the grid is fine enough to perform an underresolved DNS, the turbulence model is used to improve the results.

The first situation is given if the Reynolds number is very large compared with the grid size. This situation is more common in applications and it is considered here.

We studied the mean velocity profile, the rms turbulence intensities  $u_{\text{rms}}^{h,*}$ ,  $v_{\text{rms}}^{h,*}$ ,  $w_{\text{rms}}^{h,*}$  and the off-diagonal Reynolds stress component  $\mathbb{R}_{12}^{h,*}$ . Qualitatively the same results were obtained for both time steps and we present results for the smaller one.

Representative examples comparing the implicit and explicit discretization of the projection terms are given in Figure 1. Whereas the differences in the curves are small for the turbulent viscosity defined with all resolved scales, the curves obtained with the combination EXPL\_SMALL differ considerably from their implicit counterparts. This behavior is caused by the strong fluctuations of the resolved small scales: it makes a big difference if these scales are taken from the previous or the present discrete time. This high sensitivity of the projection terms on the temporal discretization is not desirable. We will consider below only the implicit approach.

Differences from the mean velocity profile for all variants of the implicit approach are shown in Figure 2. The deviation from the reference curve is in general smaller for  $C_s = 0.01$  and the results on Grid 1 are better than on the other grids. The curves obtained with the two models for  $v_T$  are in general similar.

Results for second-order statistics are given in Figure 3. All simulations overpredict the (absolute) values of the reference curves. The rather large differences to the reference curves are caused by the used coarse grid. We checked that the differences become smaller on finer grids (which allow to perform an underresolved DNS). The overprediction is often smaller for  $v_T$  defined with all resolved scales, if all other parameters of the simulations were chosen to be the same. The results obtained with  $C_s = 0.01$  are in general closer to the reference curves than the results with

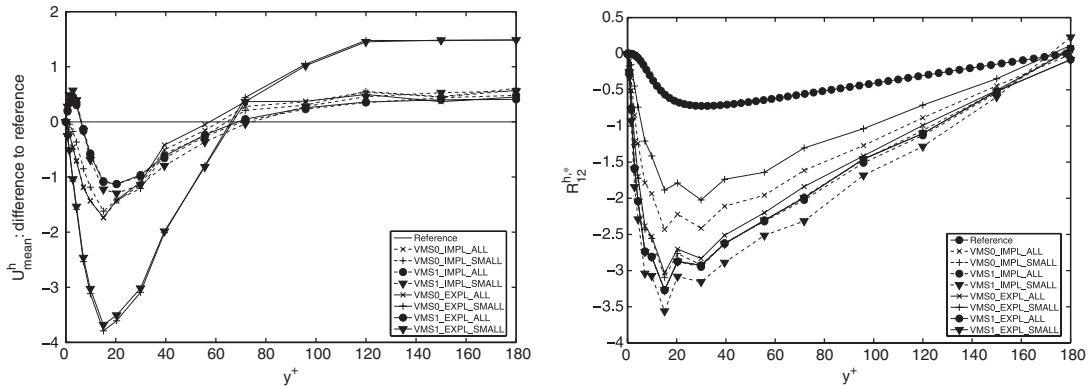


Figure 1. Comparison of the implicit and explicit discretization of the projection terms; left: difference from the mean velocity profile ( $U_{\max} = 18.301$  at  $y^+ = 180$ ), Grid 1,  $C_s = 0.01$  and right:  $R_{12}^{h,*}$ , Grid 2,  $C_s = 0.005$ .

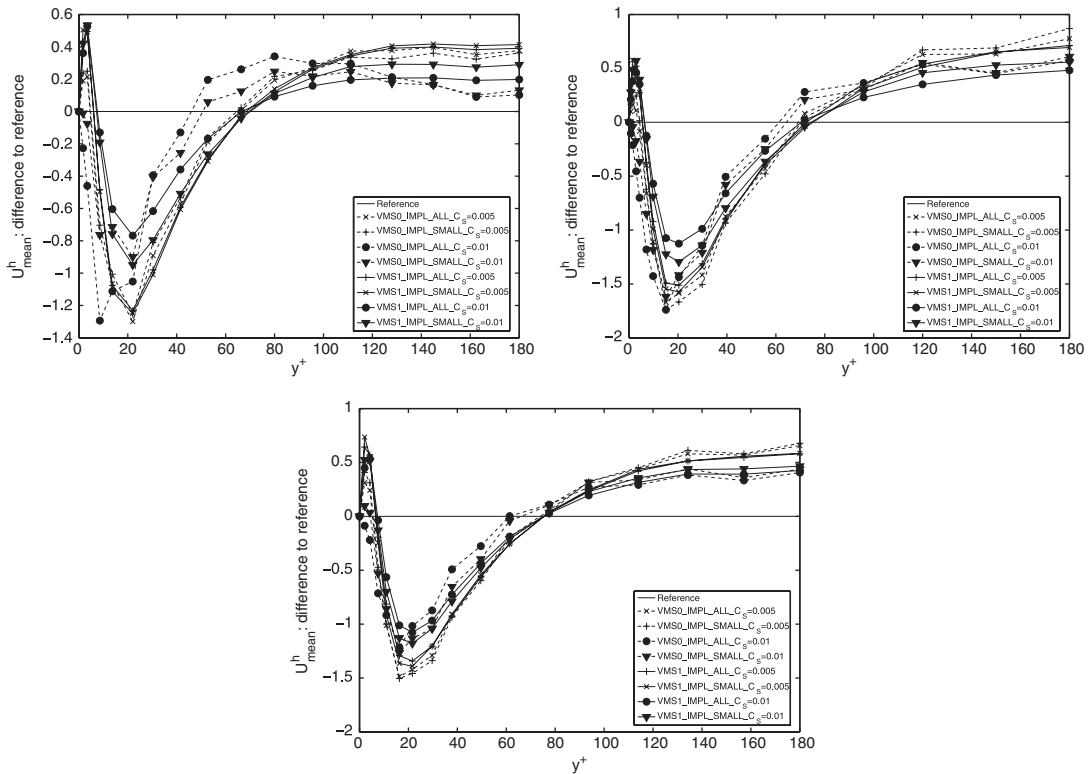


Figure 2. Differences from the reference mean velocity profile ( $U_{\max} = 18.301$  at  $y^+ = 180$ ), Grid 1 (left), Grid 2 (right), and Grid 3 (bottom).

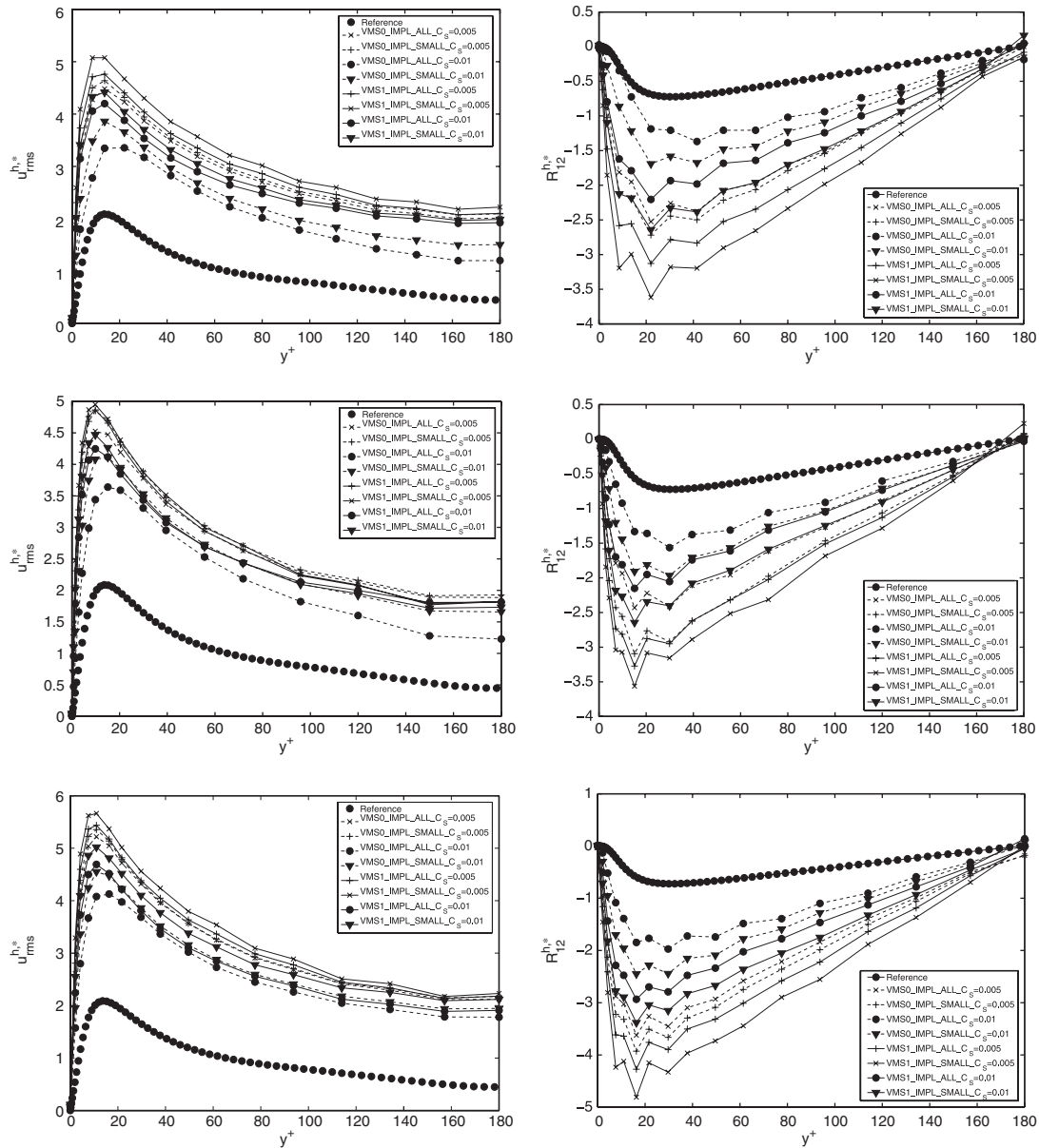


Figure 3. Second-order statistics:  $u_{rms}^{h,*}$  (left),  $R_{12}^{h,*}$  (right); Grid 1, Grid 2, Grid 3 (top to bottom).

$C_s = 0.005$  and VMS0 leads often to better results than VMS1. Thus, the VMS method, which introduces most turbulent viscosity, VMS0\_ALL with  $C_s = 0.01$ , shows the smallest overprediction of the second-order statistics. Comparing the grids, the worst results are always obtained on Grid 3.

Table I. Computing times in seconds on Grid 1 for a period of 10 s (5000 time steps).

	IMPL_ALL	IMPL_SMALL	EXPL_ALL	EXPL_SMALL
VMS0, $C_s = 0.005$	309 394	315 292	286 341	277 900
VMS1, $C_s = 0.005$	363 972	384 424	340 980	294 674
VMS0, $C_s = 0.01$	228 071	269 520	217 644	203 413
VMS1, $C_s = 0.01$	316 117	333 937	288 845	208 472

Computing times on Grid 1 are given in Table I. The same solver as in [10] was used. For using all resolved scales in the definition of  $v_T$ , the explicit approach saved consistently 5–9% of the computing time compared with the implicit one. The larger differences in the results of both temporal discretizations for the resolved small scales in the definition of  $v_T$  are reflected by larger differences in the computing times. We consider the results with IMPL\_SMALL more reliable than with EXPL\_SMALL, which is supported for instance with the left picture of Figure 1. The computations with IMPL\_SMALL took somewhat more time than the implicit approach with all resolved scales in the definition of  $v_T$ .

#### 4. SUMMARY

This paper investigated variations of parameters in a three-scale projection-based finite element VMS method with respect to the large-scale space, the form of the turbulent viscosity and the temporal discretization of the projection terms. Three different grids commonly used in turbulent flow computations were studied. The computations were performed on rather coarse grids.

The main observation was that the results obtained with the resolved small scales in the definition of  $v_T$  are sensitive to the temporal discretization of the projection terms. Using these scales from different times may lead to much different results because of their strong fluctuations. The second-order statistics were overpredicted in all simulations. The method that introduces most viscosity (VMS0\_ALL with  $C_s = 0.01$ ) often gave the best results. The results on Grid 3 are often worse than on the other grids. The explicit approach with all resolved scales in  $v_T$  was somewhat (always less than 10%) faster than the corresponding implicit approach.

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